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(54) **MULTI-STAGE TRANSMISSION FOR VEHICLE**

(56) **References Cited**

U.S. PATENT DOCUMENTS

8,303,455	B2	11/2012	Gumpoltsberger et al.	
2002/0119859	A1	8/2002	Raghavan et al.	
2008/0103015	A1 *	5/2008	Seo	475/280
2010/0048344	A1	2/2010	Kamm et al.	
2010/0069195	A1 *	3/2010	Baldwin	475/276
2015/0094185	A1 *	4/2015	Beck et al.	475/275
2015/0119188	A1 *	4/2015	Beck et al.	475/276

FOREIGN PATENT DOCUMENTS

KR	2011-0131816 (A)	12/2011
KR	2013-0031455 (A)	3/2013
KR	2013-0077146 (A)	7/2013

* cited by examiner

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(57) **ABSTRACT**

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F16H 3/62 (2006.01)
F16H 3/66 (2006.01)

(52) **U.S. Cl.**
CPC **F16H 3/66** (2013.01); **F16H 2200/0069** (2013.01); **F16H 2200/2012** (2013.01); **F16H 2200/2046** (2013.01)

(58) **Field of Classification Search**
CPC F16H 3/62; F16H 3/66; F16H 2200/0069; F16H 2200/2012; F16H 2200/2046
See application file for complete search history.

A multi-stage transmission for a vehicle may include an input shaft and an output shaft, a first planetary gear device, a second planetary gear device, a third planetary gear device, and a fourth planetary gear device provided between the input shaft and the output shaft to transfer a torque and each including three rotating elements, and at least six shifting elements connected to the rotating elements of the first to fourth planetary gear devices.

7 Claims, 2 Drawing Sheets

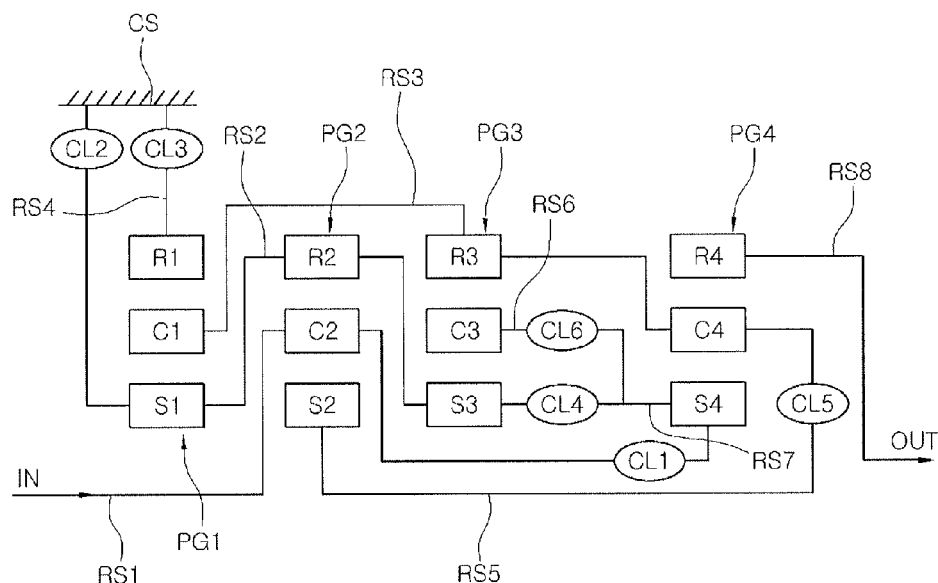


FIG. 1

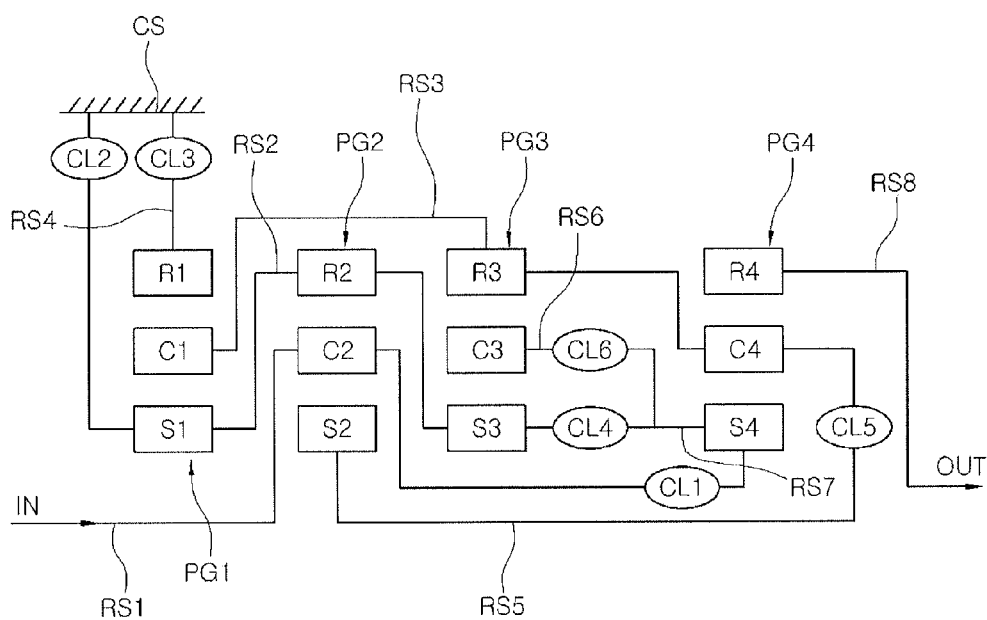


FIG. 2

SHIFTING POSITION	CL1	CL2	CL3	CL4	CL5	CL6	GEAR RATIO
1ST	○		○	○			5.909
2ND			○	○	○		4.727
3RD	○		○		○		3.250
4TH			○		○	○	2.476
5TH	○		○			○	1.857
6TH	○			○		○	1.000
7TH	○	○				○	0.591
8TH		○			○	○	0.295
9TH	○	○			○		0.265
10TH		○		○	○		0.241
REV	○	○	○				2.600

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MULTI-STAGE TRANSMISSION FOR VEHICLE

CROSS REFERENCE TO RELATED APPLICATION

The present application claims priority to Korean Patent Application No. 10-2014-0166839, filed Nov. 26, 2014, the entire contents of which is incorporated herein for all purposes by this reference.

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present invention relates to a multi-stage transmission for a vehicle, and more particularly, to a technology capable of improving fuel efficiency of a vehicle by implementing as many shifting stages as possible by as few and small components as possible and a simple configuration.

2. Description of Related Art

Recently, a rise in an oil prices has become a factor allowing automobile manufacturers throughout the world to enter into unlimited competition toward fuel efficiency improvement, and in the case of an engine, an effort to improve fuel efficiency and decrease a weight through a technology such as downsizing, or the like, has been conducted.

Meanwhile, among methods of improving fuel efficiency that may be made by a transmission mounted in a vehicle, there is a method of allowing an engine to be driven at a more efficient driving point through a multi-stage transmission to ultimately improve fuel efficiency.

In addition, the multi-stage transmission as described above may allow the engine to be driven in a relatively low revolution per minute (RPM) band to further improve silence of the vehicle.

However, as shifting stages of the transmission are increased, the number of internal components configuring the transmission is increased, such that a mounting feature and transfer efficiency may be deteriorated and a cost and a weight may be increased. Therefore, in order to maximize a fuel efficiency improving effect through the multi-stage transmission, it is important to devise a transmission structure capable of deriving maximum efficiency by a small number of components and a comparatively simple configuration.

The information disclosed in this Background of the Invention section is only for enhancement of understanding of the general background of the invention and should not be taken as an acknowledgement or any form of suggestion that this information forms the prior art already known to a person skilled in the art.

BRIEF SUMMARY

Various aspects of the present invention are directed to providing a multi-stage transmission for a vehicle capable of maximizing fuel efficiency improvement of the vehicle through driving at an optimal driving point of an engine and improving silence of the vehicle through more silent driving of the engine by implementing at least forward ten stages and reverse one stage or more by a comparatively small number of components and a simple configuration.

According to various aspects of the present invention, a multi-stage transmission for a vehicle, may include an input shaft and an output shaft, a first planetary gear device, a second planetary gear device, a third planetary gear device, and a fourth planetary gear device provided between the input shaft and the output shaft to transfer a torque and each includ-

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ing three rotating elements, and at least six shifting elements connected to the rotating elements of the first to fourth planetary gear devices, in which a first rotating element of the first planetary gear device may be continuously connected to a third rotating element of the second planetary gear device and may be fixably installed by any one of the shifting elements, a second rotating element of the first planetary gear device may be continuously connected to a third rotating element of the third planetary gear device, and a third rotating element of the first planetary gear device may be fixably installed by another of the shifting elements, a first rotating element of the second planetary gear device may be selectively connected to a second rotating element of the fourth planetary gear device, a second rotating element of the second planetary gear device may be continuously connected to the input shaft and selectively connected to a first rotating element of the fourth planetary gear device, and the third rotating element of the second planetary gear device may be continuously connected to a first rotating element of the third planetary gear device, each of the first and second rotating elements of the third planetary gear device may be selectively connected to the first rotating element of the fourth planetary gear device and the third rotating element of the third planetary gear device may be continuously connected to the second rotating element of the fourth planetary gear device, and a third rotating element of the fourth planetary gear device may be continuously connected to the output shaft.

The first planetary gear device, the second planetary gear device, the third planetary gear device, and the fourth planetary gear device may be sequentially disposed in an axial direction of the input shaft and the output shaft.

The first rotating element of the first planetary gear device may be fixably installed to a transmission case by a second clutch among the shifting elements, the third rotating element of the first planetary gear device may be fixably installed to the transmission case by a third clutch among the shifting elements, and the remaining shifting elements may form selective connection structures between the rotating elements of the planetary gear devices.

A first clutch among the shifting elements may form the selective connection structure between the second rotating element of the second planetary gear device and the first rotating element of the fourth planetary gear device, a fourth clutch among the shifting elements may form the selective connection structure between the first rotating element of the third planetary gear device and the first rotating element of the fourth planetary gear device, a fifth clutch among the shifting elements may form the selective connection structure between the first rotating element of the second planetary gear device and the second rotating element of the fourth planetary gear device, and a sixth clutch among the shifting elements may form the selective connection structure between the second rotating element of the third planetary gear device and the first rotating element of the fourth planetary gear device.

According to various aspects of the present invention, a multi-stage transmission for a vehicle may include a first planetary gear device, a second planetary gear device, a third planetary gear device, and a fourth planetary gear device each having three rotating elements, six shifting elements configured to selectively provide frictional force, and eight shafts connected to the rotating elements of the planetary gear devices, in which a first shaft may be an input shaft connected in series with a second rotating element of the second planetary gear device, a second shaft may be connected in series with a first rotating element of the first planetary gear device, a third rotating element of the second planetary gear device,

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and a first rotating element of the third planetary gear device, a third shaft may be connected in series with a second rotating element of the first planetary gear device, a third rotating element of the third planetary gear device, and a second rotating element of the fourth planetary gear device, a fourth shaft may be connected in series with a third rotating element of the first planetary gear device, a fifth shaft may be connected in series with a first rotating element of the second planetary gear device, a sixth shaft may be connected in series with a second rotating element of the third planetary gear device, a seventh shaft may be connected in series with a first rotating element of the fourth planetary gear device, and an eighth shaft may be an output shaft connected in series with a third rotating element of the fourth planetary gear device, in which a first clutch among the six shifting elements may be installed between the first shaft and the seventh shaft, a second clutch among the six shifting elements may be installed between the second shaft and a transmission case, a third clutch among the six shifting elements may be installed between the fourth shaft and the transmission case, a fourth clutch among the six shifting elements may be installed between the second shaft and the seventh shaft, a fifth clutch among the six shifting elements may be installed between the third shaft and the fifth shaft, and a sixth clutch among the six shifting elements may be installed between the sixth shaft and the seventh shaft.

It is understood that the term “vehicle” or “vehicular” or other similar terms as used herein is inclusive of motor vehicles in general such as passenger automobiles including sports utility vehicles (SUV), buses, trucks, various commercial vehicles, watercraft including a variety of boats and ships, aircraft, and the like, and includes hybrid vehicles, electric vehicles, plug-in hybrid electric vehicles, hydrogen-powered vehicles and other alternative fuel vehicles (e.g., fuel derived from resources other than petroleum). As referred to herein, a hybrid vehicle is a vehicle that has two or more sources of power, for example, both gasoline-powered and electric-powered vehicles.

The methods and apparatuses of the present invention have other features and advantages which will be apparent from or are set forth in more detail in the accompanying drawings, which are incorporated herein, and the following Detailed Description, which together serve to explain certain principles of the present invention.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a configuration diagram of an exemplary multi-stage transmission for a vehicle according to an exemplary embodiment of the present invention.

FIG. 2 is a table showing operation modes of the exemplary multi-stage transmission for the vehicle of FIG. 1.

It should be understood that the appended drawings are not necessarily to scale, presenting a somewhat simplified representation of various features illustrative of the basic principles of the invention. The specific design features of the present invention as disclosed herein, including, for example, specific dimensions, orientations, locations, and shapes will be determined in part by the particular intended application and use environment.

DETAILED DESCRIPTION

Reference will now be made in detail to various embodiments of the present invention(s), examples of which are illustrated in the accompanying drawings and described below. While the invention(s) will be described in conjunc-

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tion with exemplary embodiments, it will be understood that the present description is not intended to limit the invention(s) to those exemplary embodiments. On the contrary, the invention(s) is/are intended to cover not only the exemplary embodiments, but also various alternatives, modifications, equivalents and other embodiments, which may be included within the spirit and scope of the invention as defined by the appended claims.

Referring to FIG. 1 and FIG. 2, a multi-stage transmission for a vehicle according to various embodiments of the present invention includes an input shaft IN and an output shaft OUT, a first planetary gear device PG1, a second planetary gear device PG2, a third planetary gear device PG3, and a fourth planetary gear device PG4 provided between the input shaft IN and the output shaft OUT so as to transfer a torque and each including three rotating elements; and at least six shifting elements connected to the rotating elements of the first to fourth planetary gear devices.

A first rotating element S1 of the first planetary gear device PG1 is continuously connected to a third rotating element R2 of the second planetary gear device PG2 and is fixably installed by any one of the shifting elements, a second rotating element C1 thereof is continuously connected to a third rotating element R3 of the third planetary gear device PG3, and a third rotating element R1 thereof is fixably installed by another of the shifting elements.

A first rotating element S2 of the second planetary gear device PG2 is selectively connected to a second rotating element C4 of the fourth planetary gear device PG4, a second rotating element C2 thereof is continuously connected to the input shaft IN and is selectively connected to a first rotating element S4 of the fourth planetary gear device PG4, and the third rotating element R2 thereof is continuously connected to a first rotating element S3 of the third planetary gear device PG3.

Each of the first and second rotating elements S3 and C3 of the third planetary gear device PG3 is selectively connected to the first rotating element S4 of the fourth planetary gear device PG4 and the third rotating element R3 thereof is continuously connected to the second rotating element C4 of the planetary gear device PG4, and a third rotating element R4 of the planetary gear device PG4 is continuously connected to the output shaft OUT.

The first planetary gear device PG1, the second planetary gear device PG2, the third planetary gear device PG3, and the fourth planetary gear device PG4 are sequentially disposed in an axial direction of the input shaft IN and the output shaft OUT.

The first rotating element S1 of the first planetary gear device PG1 is fixably installed to a transmission case CS by a second clutch CL2 among the shifting elements; and the third rotating element R1 of the first planetary gear device PG1 is fixably installed to the transmission case CS by a third clutch CL3 among the shifting elements.

Therefore, the second clutch CL2 and the third clutch CL3 serve as brakes, respectively, to serve to restrict or allow rotation of the first rotating element S1 and the third rotating element R1 of the first planetary gear device PG1, respectively.

The others among the shifting elements are configured so as to form selective connection structures between the rotating elements of the planetary gear devices.

That is, a first clutch CL1 among the shifting elements forms the selective connection structure between the second rotating element C2 of the second planetary gear device PG2 and the first rotating element S4 of the fourth planetary gear device PG4, a fourth clutch CL4 among the shifting elements

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forms the selective connection structure between the first rotating element S3 of the third planetary gear device PG3 and the first rotating element S4 of the fourth planetary gear device PG4, a fifth clutch CL5 among the shifting elements forms the selective connection structure between the first rotating element S2 of the second planetary gear device PG2 and the second rotating element C4 of the fourth planetary gear device PG4; and a sixth clutch CL6 among the shifting elements forms the selective connection structure between the second rotating element C3 of the third planetary gear device PG3 and the first rotating element S4 of the fourth planetary gear device PG4.

In various embodiments, the first rotating element S1, the second rotating element C1, and the third rotating element R1 of the first planetary gear device PG1 are a first sun gear, a first carrier, and a first ring gear, respectively, the first rotating element S2, the second rotating element C2, and the third rotating element R2 of the second planetary gear device PG2 are a second sun gear, a second carrier, and a second ring gear, respectively, the first rotating element S3, the second rotating element C3, and the third rotating element R3 of the third planetary gear device PG3 are a third sun gear, a third carrier, and a third ring gear, respectively, and the first rotating element S4, the second rotating element C4, and the third rotating element R4 of the fourth planetary gear device PG4 are a fourth sun gear, a fourth carrier, and a fourth ring gear, respectively.

The multi-stage transmission for a vehicle configured as described above may also be represented as follows.

That is, the multi-stage transmission for a vehicle according to various embodiments of the present invention is configured to include the first planetary gear device PG1, the second planetary gear device PG2, the third planetary gear device PG3, and the fourth planetary gear device PG4 each having three rotating elements, the six shifting elements configured so as to selectively provide frictional force, and eight shafts connected to the rotating elements of the planetary gear devices.

Here, a first shaft RS1 is the input shaft IN connected in series with the second rotating element C2 of the second planetary gear device PG2, a second shaft RS2 is connected in series with the first rotating element S1 of the first planetary gear device PG1, the third rotating element R2 of the second planetary gear device PG2, and the first rotating element S3 of the third planetary gear device PG3, a third shaft RS3 is connected in series with the second rotating element C1 of the first planetary gear device PG1, the third rotating element R3 of the third planetary gear device PG3, and the second rotating element C4 of the fourth planetary gear device PG4, a fourth shaft RS4 is connected in series with the third rotating element R1 of the first planetary gear device PG1, a fifth shaft RS5 is connected in series with the first rotating element S2 of the second planetary gear device PG2, a sixth shaft RS6 is connected in series with the second rotating element C3 of the third planetary gear device PG3, a seventh shaft RS7 is connected in series with the first rotating element S4 of the fourth planetary gear device PG4, and an eighth shaft RS8 is the output shaft OUT connected in series with the third rotating element R4 of the fourth planetary gear device PG4.

In addition, the first clutch CL1 among the six shifting elements is installed between the first shaft RS1 and the seventh shaft RS7, the second clutch CL2 among the six shifting elements is installed between the second shaft RS2 and the transmission case CS, the third clutch CL3 among the six shifting elements is installed between the fourth shaft RS4 and the transmission case CS, the fourth clutch CL4 among the six shifting elements is installed between the second shaft

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RS2 and the seventh shaft RS7, the fifth clutch CL5 among the six shifting elements is installed between the third shaft RS3 and the fifth shaft RS5, and the sixth clutch CL6 among the six shifting elements is installed between the sixth shaft RS6 and the seventh shaft RS7.

Since the multi-stage transmission for a vehicle according to various embodiments of the present invention including the four simple planetary gear devices and the six shifting elements as described above may implement at least ten forward stages and one reverse stage depending on an operation mode table as shown in FIG. 2, it may implement a multi-stage shifting stage of at least ten stages by a relatively small number of components and a simple configuration to contribute to improvement of fuel efficiency and silence of the vehicle, thereby making it possible to improve salability of the vehicle.

As set forth above, according to various embodiments of the present invention, at least ten forward stages and one reverse stage or more are implemented by a comparatively small number of components and a simple configuration, thereby making it possible to maximize fuel efficiency improvement of the vehicle through driving at an optimal driving point of an engine and improve silence of the vehicle through more silent driving of the engine.

The foregoing descriptions of specific exemplary embodiments of the present invention have been presented for purposes of illustration and description. They are not intended to be exhaustive or to limit the invention to the precise forms disclosed, and obviously many modifications and variations are possible in light of the above teachings. The exemplary embodiments were chosen and described in order to explain certain principles of the invention and their practical application, to thereby enable others skilled in the art to make and utilize various exemplary embodiments of the present invention, as well as various alternatives and modifications thereof. It is intended that the scope of the invention be defined by the Claims appended hereto and their equivalents.

What is claimed is:

1. A multi-stage transmission for a vehicle, comprising:
 - an input shaft and an output shaft;
 - a first planetary gear device, a second planetary gear device, a third planetary gear device, and a fourth planetary gear device provided between the input shaft and the output shaft to transfer a torque and each of the first to fourth planetary gear devices including three rotating elements; and
 - six shifting elements, wherein each of the six shifting elements is selectively engaged with at least one of the rotating elements of the first to fourth planetary gear devices,
- wherein a first rotating element of the first planetary gear device is continuously connected to a third rotating element of the second planetary gear device and is selectively engaged with one of the six shifting elements, a second rotating element of the first planetary gear device is continuously connected to a third rotating element of the third planetary gear device, and a third rotating element of the first planetary gear device is selectively engaged with another one of the six shifting elements,
- a first rotating element of the second planetary gear device is selectively connected to a second rotating element of the fourth planetary gear device, a second rotating element of the second planetary gear device is continuously connected to the input shaft and is selectively connected to a first rotating element of the fourth planetary gear device, and the third rotating element of the second

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planetary gear device is continuously connected to a first rotating element of the third planetary gear device, each of the first and second rotating elements of the third planetary gear device is selectively connected to the first rotating element of the fourth planetary gear device and the third rotating element of the third planetary gear device is continuously connected to the second rotating element of the fourth planetary gear device, and a third rotating element of the fourth planetary gear device is continuously connected to the output shaft.

2. The multi-stage transmission for the vehicle of claim 1, wherein the first planetary gear device, the second planetary gear device, the third planetary gear device, and the fourth planetary gear device are sequentially disposed in an axial direction of the input shaft and the output shaft.

3. The multi-stage transmission for the vehicle of claim 2, wherein the first rotating element of the first planetary gear device is selectively engaged to a transmission case by a second clutch among the six shifting elements,

the third rotating element of the first planetary gear device is selectively engaged to the transmission case by a third clutch among the six shifting elements, and

the remaining shifting elements form selective connection structures between the rotating elements of the first, second, and third planetary gear devices.

4. The multi-stage transmission for the vehicle of claim 3, wherein a first clutch among the six shifting elements forms the selective connection structure between the second rotating element of the second planetary gear device and the first rotating element of the fourth planetary gear device,

a fourth clutch among the six shifting elements forms the selective connection structure between the first rotating element of the third planetary gear device and the first rotating element of the fourth planetary gear device,

a fifth clutch among the six shifting elements forms the selective connection structure between the first rotating element of the second planetary gear device and the second rotating element of the fourth planetary gear device, and

a sixth clutch among the six shifting elements forms the selective connection structure between the second rotating element of the third planetary gear device and the first rotating element of the fourth planetary gear device.

5. A multi-stage transmission for a vehicle, comprising: a first planetary gear device, a second planetary gear device, a third planetary gear device, and a fourth planetary gear device each having three rotating elements; six shifting elements configured to selectively provide frictional force; and

eight shafts, wherein each of the eight shafts is connected with at least one of the rotating elements of the first to fourth planetary gear devices,

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wherein a first shaft is an input shaft connected in series with a second rotating element of the second planetary gear device,

a second shaft is connected in series with a first rotating element of the first planetary gear device, a third rotating element of the second planetary gear device, and a first rotating element of the third planetary gear device,

a third shaft is connected in series with a second rotating element of the first planetary gear device, a third rotating element of the third planetary gear device, and a second rotating element of the fourth planetary gear device,

a fourth shaft is connected in series with a third rotating element of the first planetary gear device,

a fifth shaft is connected in series with a first rotating element of the second planetary gear device,

a sixth shaft is connected in series with a second rotating element of the third planetary gear device,

a seventh shaft is connected in series with a first rotating element of the fourth planetary gear device, and

an eighth shaft is an output shaft connected in series with a third rotating element of the fourth planetary gear device,

wherein a first clutch among the six shifting elements is installed between the first shaft and the seventh shaft, a second clutch among the six shifting elements is installed between the second shaft and a transmission case, a third clutch among the six shifting elements is installed between the fourth shaft and the transmission case, a fourth clutch among the six shifting elements is installed between the second shaft and the seventh shaft, a fifth clutch among the six shifting elements is installed between the third shaft and the fifth shaft, and a sixth clutch among the six shifting elements is installed between the sixth shaft and the seventh shaft.

6. The multi-stage transmission for the vehicle of claim 5, wherein the first planetary gear device, the second planetary gear device, the third planetary gear device, and the fourth planetary gear device are sequentially disposed in an axial direction of the input shaft and the output shaft.

7. The multi-stage transmission for the vehicle of claim 5, wherein the first clutch selectively connects the second rotating element of the second planetary gear device and the first rotating element of the fourth planetary gear device,

the fourth clutch selectively connects the first rotating element of the third planetary gear device and the first rotating element of the fourth planetary gear device,

the fifth clutch selectively connects the first rotating element of the second planetary gear device and the second rotating element of the fourth planetary gear device, and

the sixth clutch selectively connects the second rotating element of the third planetary gear device and the first rotating element of the fourth planetary gear device.

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